

Fall, 2022

# ME 323 – Mechanics of Materials

## Lecture 17 – Deflection of beams (cont.)

Reading assignment: Ch.9, Ch.11 lecturebook



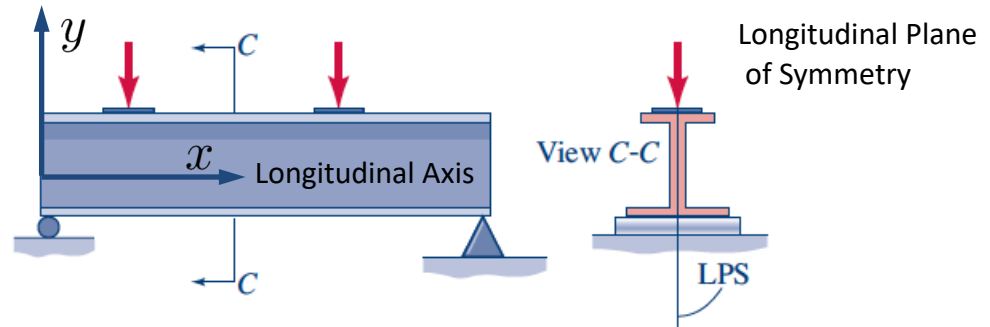
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# Deflection of beams

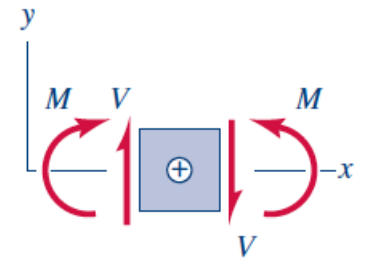
## Beam theory (@ ME 323)

- Geometry of the solid body: straight, slender member with constant cross section that is design to support transverse loads.

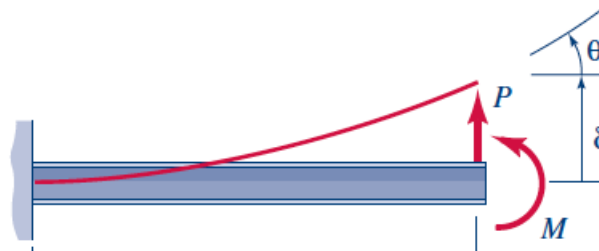


- Kinematic assumptions: Bernoulli-Euler Beam Theory
- Material behavior: isotropic linear elastic material; small deformations.

- Equilibrium:
  - 1) relate stress distribution (normal and shear stress) with internal resultants (only shear and bending moment)



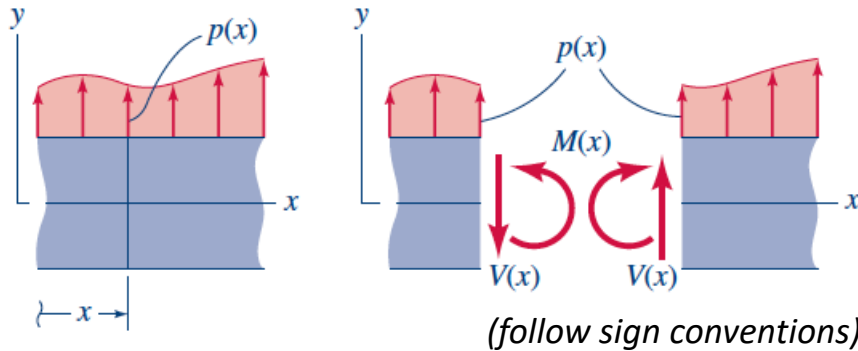
- 2) find deformed configuration



# Deflection of beams

## Load-deflection equations

$$V'(x) = p(x) \quad M'(x) = V(x)$$



$$M(x) = E(x)I(x)v''(x)$$

Moment-curvature eqn.

$$V(x) = (E(x)I(x)v''(x))'$$

Shear-deflection eqn.

$$p(x) = (E(x)I(x)v''(x))''$$

Load-deflection eqn.

(constant cross-section and material properties)

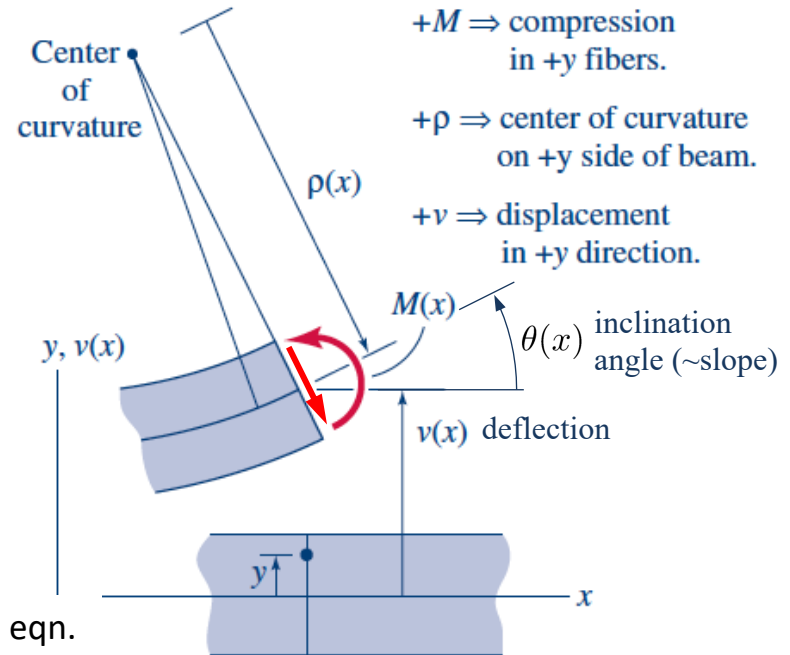
$$M = EIv''$$

(2<sup>nd</sup> order)

$$V = EIv'''$$

$$p = EIv''''$$

(4<sup>th</sup> order)



+M  $\Rightarrow$  compression in +y fibers.

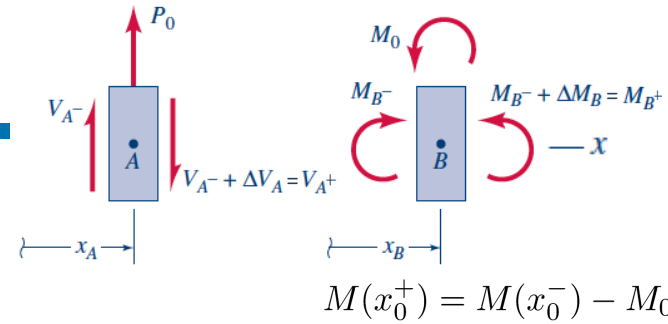
+ $\rho$   $\Rightarrow$  center of curvature on +y side of beam.

+v  $\Rightarrow$  displacement in +y direction.

$$\theta(x) \approx \frac{dv(x)}{dx}$$

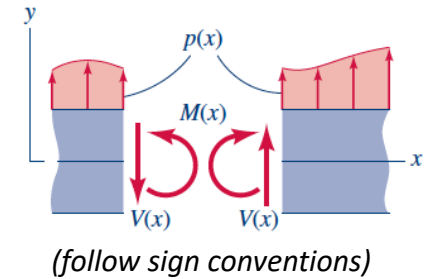
$$\frac{1}{\rho(x)} \approx \frac{d^2v(x)}{dx^2}$$

# Deflection of beams

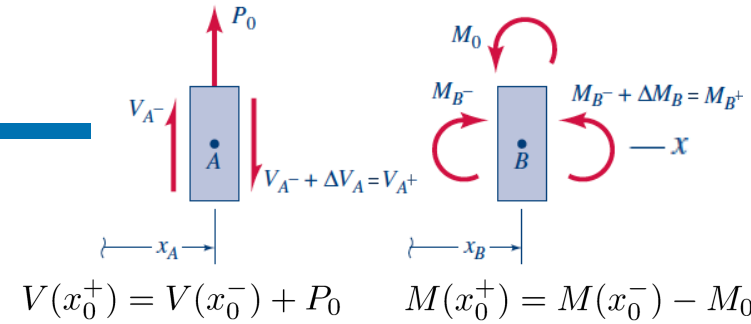


## Boundary conditions

	Type	Symbol*	2nd Order	4th Order
<b>BC</b>	Fixed end		$v = 0$ $v' = 0$	$v = 0$ $v' = 0$
	Simple support		$v = 0$	$v = 0$ $M = 0$
	Free end		No BC	$V = 0$ $M = 0$
	Concentrated force		No BC	$V = P_0$ $M = 0$
	Concentrated couple		No BC	$V = 0$ $M = -M_0$
	Constrained rotation end			$v' = 0$



# Deflection of beams



## Continuity conditions

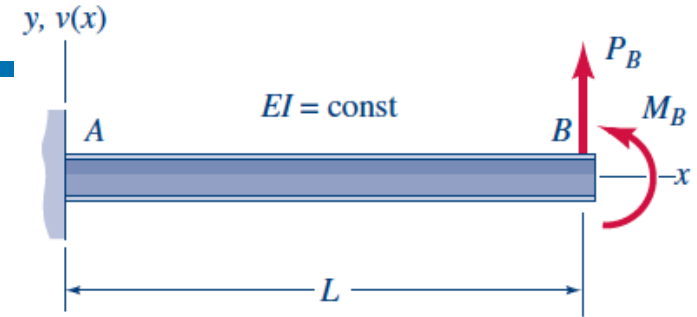
	Type	Symbol	2nd Order	4th Order
CC	Roller		$v_1 = v_2 = 0$ $v'_1 = v'_2$	$v_1 = v_2 = 0$ $v'_1 = v'_2$ $M_1 = M_2$
	Discontinuity in load function		$v_1 = v_2$ $v'_1 = v'_2$	$v_1 = v_2$ $v'_1 = v'_2$ $V_1 = V_2$ $M_1 = M_2$
	Concentrated force		$v_1 = v_2$ $v'_1 = v'_2$	$v_1 = v_2, v'_1 = v'_2$ $V_2 - V_1 = P_0$ $M_1 = M_2$
	Concentrated couple		$v_1 = v_2$ $v'_1 = v'_2$	$v_1 = v_2, v'_1 - v'_2$ $V_1 = V_2$ $M_2 - M_1 = -M_0$
	Pin, with force		$v_1 = v_2$	$v_1 = v_2$ $V_2 - V_1 = P_0$ $M_1 = M_2 = 0$

# Deflection of beams

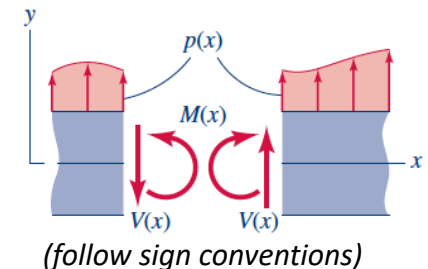
## Example 30:

The beam shown in the figure is completely fixed at end A. Determine an expression for the deflection curve  $v(x)$ , and draw the shear force and bending moment diagrams.

Use the second-order method.



$$V(x_0^+) = V(x_0^-) + P_0$$
$$M(x_0^+) = M(x_0^-) - M_0$$

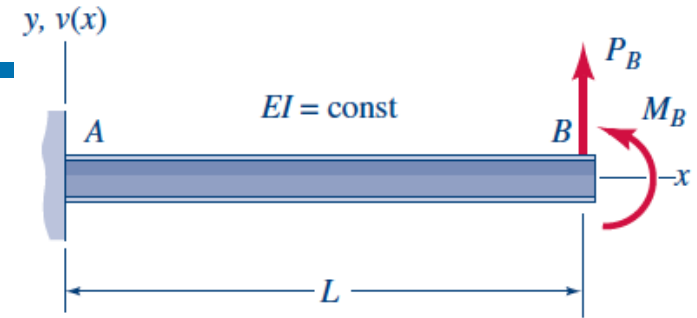


# Deflection of beams

## Example 31 (Practice problem):

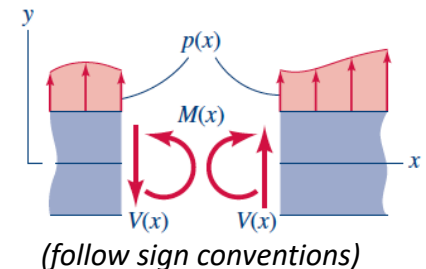
The beam shown in the figure is completely fixed at end A. Determine an expression for the deflection curve  $v(x)$ .

Use the fourth-order method (4 unknowns, 4 B.C.).

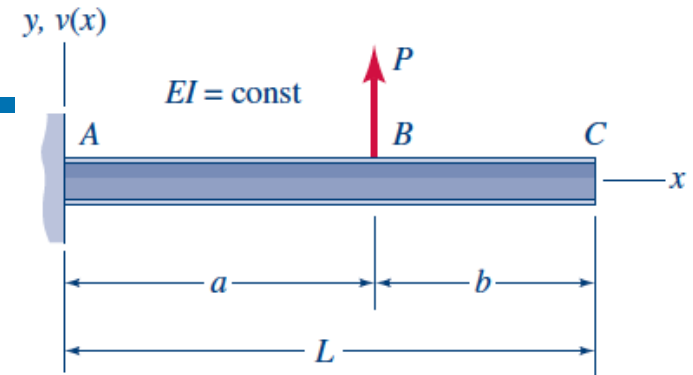


$$V(x_0^+) = V(x_0^-) + P_0$$

$$M(x_0^+) = M(x_0^-) - M_0$$

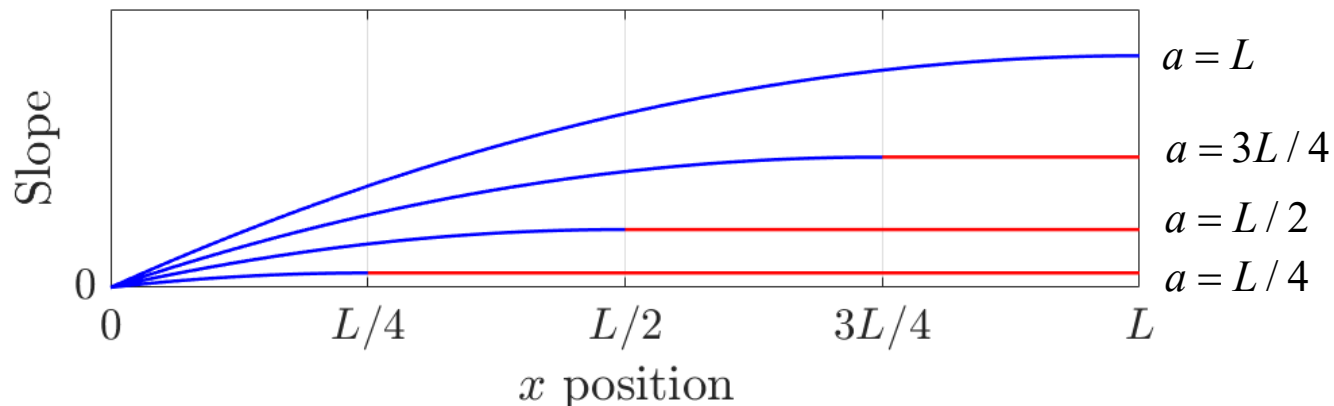
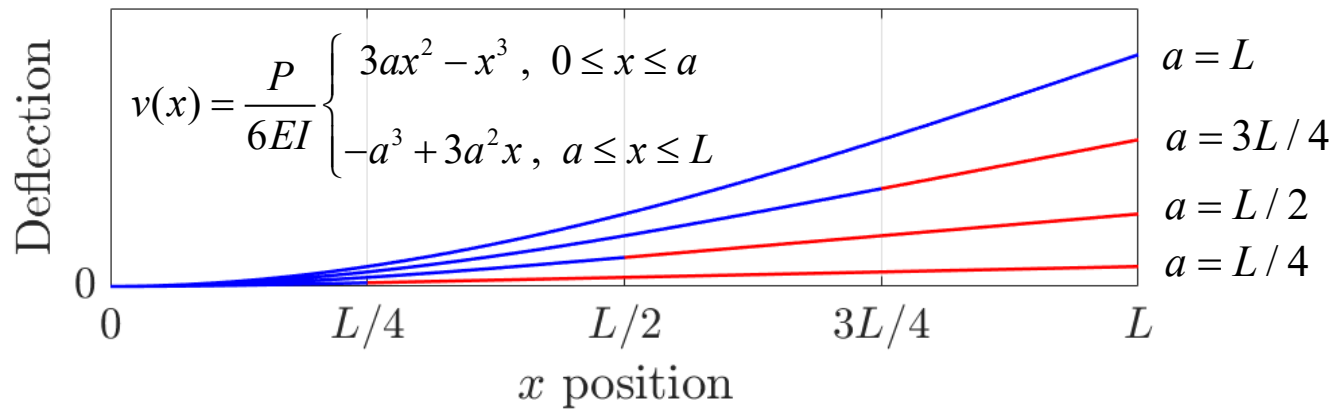


# Deflection of beams



## Example 32:

The beam shown in the figure is completely fixed at end A. Determine an expression for the deflection curve  $v(x)$ .





# Deflection of beams – Indeterminate problems

Outline for 2<sup>nd</sup> order method (determinate or indeterminate):

- FBD
- Equilibrium for external forces and couples
- Find internal moment  $M(x)$  for each section
- Integrate moment-curvature equation  $EIv''(x) = M(x)$
- Apply boundary and continuity conditions
- Solve for unknowns
- Check units!
- Draw (i) deflection curve, (ii) shear force diagram, and (iii) bending moment diagram

# Deflection of beams

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Any questions?